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PATENT APPLICATION

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TROUGHING IDLERS FOR BELT CONVEYORS

TECHNICAL FIELD OF THE INVENTION

5 The present invention relates generally to belt conveyors and, more particularly, to troughing idlers that are particularly well-suited for use in enclosed belt conveyors.

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BACKGROUND OF THE INVENTION

Hansen Manufacturing Corp. Of Sioux Falls, South
Dakota, makes and sells enclosed belt conveyors under the
trademark HI ROLLER™, in which the belt and the elements
that support it are completely enclosed in a housing, which
is usually called the "trunking." The trunking provides a
structural support for troughing idlers, the belt and the
material being conveyed, protects the material being
conveyed by keeping out foreign substances and, in an
outside installation, rain and snow, and keeps dust and any
fugitive material that falls from the belt from entering the
environment. The previously known HI ROLLER™ conveyors,
which are in widespread use for conveying various materials,
especially grains, have troughing idlers that resemble
spools in that they have a body having a circular
cylindrical medial surface and frusto-conical side surfaces
that diverge outwardly from the medial surface. The belt
nests in the body, which shapes the belt (in cross section)
into a trough. The body is affixed to a shaft, the ends of
which pass through holes in the side walls of the conveyor
trunking and are supported in bearings located outside of
the trunking. An important advantage of the spool-like
troughing idlers is that the bearings are located outside of

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the enclosure and are, therefore, not exposed to any dust produced by the materials being conveyed and are accessible for greasing from time to time. Also, the likelihood of any grease that might leak from a bearing entering the trunking and contaminating the material being conveyed is very small. Inasmuch as the body and shaft rotate, dust cannot build up on them, and there are no internal fixed frames or supporting elements for the spool-like idlers within the troughing where dust can accumulate. Should a bearing fail and overheat, it is not likely to be a source of ignition for a fire or detonation of the dusty atmosphere within the troughing by virtue of its isolation outside of the troughing.

Spool-like troughing idlers have the disadvantage of inherently producing slippage between the belt and the frusto-conical side surfaces of the body, which run at a surface speed that progressively increases as a function of the distance radially from the circular cylindrical medial portion. The slippage causes wear of the belt and the side surfaces of the idler, especially if the material being conveyed includes hard, abrasive particles that can stick to the belt.

There are many specific forms of troughing idlers in which both a medial horizontal roller and separate sloping

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side rollers run at the same surface speeds, the rollers being of the same diameters and being mounted for rotation on separate shafts. In most designs, the rollers are tubular and supported by bearings within them that are carried by stationary shafts that are attached at opposite ends of the rollers to framing or stands. The shafts and frames are prone to buildups of dust. The bearings are close to the dusty environment of the belt, and if they fail and become highly heated, can ignite a fire. The foregoing shortcomings of previously known designs of three-roller type troughing idlers make them only marginally useful, at best, in enclosed belt conveyors, particularly for conveyors used for conveying grain where the environment is very dusty, the dust is highly flammable, and contamination is unacceptable.

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SUMMARY OF THE INVENTION

Throughout this document, the terms "inboard" and "outboard" are used for convenience to locate the positions of various components relative to other components with respect to the longitudinal center of the path of a conveyor belt supported by the troughing idlers.

One object of the present invention is to provide troughing idlers in which all running surfaces travel at equal surface velocities, thus avoiding slippage anywhere between the belt and the idlers. Another object is to avoid having exposed portions of shafts and fixed frame members supporting the shafts where dust can collect. It also an object to make the bearings that support the rollers of troughing idlers accessible for lubrication from outside trunking that fully encloses the belt and the idler rollers. Yet another object is to isolate the bearings that support troughing idler rollers as much as possible from the interior of enclosed trunking so as to make contamination of the interior of the trunking by leaking lubricant and exposure of the enclosed interior to heat from failed bearings very unlikely.

The foregoing objects are attained, in accordance with the present invention, by a troughing idler installation that includes side supports located opposite each other with

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respect to a space along which a conveyor belt moves along a belt path, a horizontal shaft extending across the space between the support members transversely to the belt path and having a medial roller affixed thereon, and antifriction bearings mounted on the side supports and supporting the horizontal shaft for rotation about a shaft axis. A side roller unit is supported exclusively by each side support member. Each side roller unit has a side roller carrier that includes a spindle, a hub, and inboard and outboard antifriction bearings interposed between the spindle and the hub, and a side roller supported in cantilevered relation with respect to the side support member by the side roller carrier for rotation about the spindle axis. The medial roller and the side rollers are arranged relative to each other so as to support the belt with side portions of the belt sloping upwardly and outwardly with respect to a horizontal medial portion of the belt.

The mounting of each side roller on the side support in cantilevered relation - each side roller is supported exclusively by a carrier unit mounted on the side support - eliminates any exposed shaft portion and a supporting frame for the side roller shaft located between the side supports in the space along which the belt runs on which dust and material that falls from the belt can collect. The

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should fail, the hub cap will retain any fragments of the bearing that might break loose and keep them from entering the belt space. Retention of fragments of failed bearings, which may be extremely hot, is of considerable importance in minimizing the chance of a failed bearing starting a fire or triggering a dust explosion in the conveyor.

Another advantage of a configuration in which the spindle is affixed to the support member and the hub is on the roller is the ability to locate the grease seal at the outboard end of the side roller carrier, which is at the highest point of the unit and will usually be outside of the space between the side supports. The high position minimizes the possibility of grease leaking from a worn or failed grease seal. The outboard location of the grease seal minimizes the possibility of leaking grease getting into the space and contaminating the material being conveyed.

The inboard and outboard antifriction bearings may be closely spaced apart and proximate to the side support member, in which case the spindle is relatively short, or they may be widely spaced apart on a spindle that extends along a major part of the length of the side roller shell. In the former case, the antifriction bearings are located as remotely as possible from the belt. A consideration here

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again is with a bearing failure and the resulting heat. Having the bearings remote from the belt and the dusty environment of the belt reduces the chance of a fire in enclosed belt conveyors used to convey grain, for example. A trade-off for a short spindle and closely spaced bearings is a greater overhung moment and higher bearing loads. A relatively longer spindle/hub results in lower bearing loads. The isolation of the bearings in the hub and a hub cap for containment of bearing fragments and exclusion of dust from the hub offers a high degree of assurance that a failed bearing will not cause a fire.

In a configuration in which a long spindle/hub and widely spaced bearings are provided, it is advantageous, for further isolation of the bearings, to provide a roller shell that is spaced apart radially from the hub and supported concentrically with the hub by annular rings. The dead air space between the hub and the roller shell is a thermal barrier to heat transfer from a failed bearing to the shell.

For economy of manufacture and minimizing bearing loads, it is advantageous to make the side roller as a tubular shell. For further thermal isolation of the bearings from a dusty environment, a dust cap should be affixed in the inboard end of the shell. When the troughing idlers of the present invention are used in an enclosed belt

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conveyor, a dust seal should be incorporated between the side roll shell or the hub and an element, such as the side support or a bracket by which the side roll carrier is mounted on the side support, to keep dust from leaking from the trunking.

Another configuration for the side roll carrier is to affix the hub to the side support member, preferably to the outboard side, and the spindle to the side roller. The spindle may have a threaded outboard end, onto which a retainer nut is threaded to retain the outboard antifriction bearing on the spindle. The hub has an outboard end located outwardly of the outboard bearing and the side roller unit further includes a hub cap on the outboard end of the hub to retain a lubricant within the hub. A grease seal is installed between the hub and the spindle inboard of the inboard antifriction bearing. The side roller has a tubular shell and a dust cap is affixed in the inboard end of the shell.

A design of the configuration just described has the advantages of simplicity and economy. Also, the roller carrier is outboard of the support and away from the dusty environment. Those advantages may not, however, outweigh an inherent large overhung moment - high bearing loads - and the location of the grease seal at the inboard side of the

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bearing where a failure can allow grease to leak into the inboard side of the side support and into the material being conveyed.

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BRIEF DESCRIPTION OF THE DRAWINGS

For a more complete understanding of the present invention and additional preferred features, and the advantages thereof, reference may be made to the following written description of exemplary embodiments, taken in conjunction with the accompanying drawings, in which:

Fig. 1 is a pictorial schematic view of a prior art enclosed belt conveyor, a portion of the trunking being broken away;

Fig. 2 is a schematic end cross-sectional view of the enclosed belt conveyor shown in Fig. 1;

Fig. 3 is an end elevational view of a trunking section of an enclosed belt conveyor that is equipped with troughing idlers according to the present invention;

Fig. 4 is a partial side elevational view of the trunking section of Fig. 3;

Fig. 5 is a side cross-sectional view, in generally schematic form, of a first embodiment of a side roller unit of a troughing idler embodying the present invention;

Fig. 6 is a side cross-sectional view, in generally schematic form, of a second embodiment of a side roller unit of a troughing idler embodying the present invention; and

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Fig. 7 is a a side cross-sectional view, in generally schematic form, of a third embodiment of a side roller unit of a troughing idler embodying the present invention.

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DESCRIPTION OF THE EMBODIMENTS

An exemplary embodiment of the present invention and its advantages are best understood by referring to FIGURES 1 to 7 of the drawings, like numerals being used for like and corresponding parts of the various FIGURES. The teachings of the present specification may be used to advantage in troughing idlers of various configurations.

A belt conveyor in which the present invention is useful to considerable advantage and which is well-known, per se, has an elongated trunking 10 that extends the entire length of the conveyor from a tail end 12 to a head end 14. The trunking 10 is essentially a hollow duct-like enclosure fabricated of steel sheet or plate stock in which all components of the conveyor other than bearings (see below), a drive motor 16, and a power transmission 18 driven by the motor are located. The trunking protects the material being conveyed from the environment and the environment from the material. Ordinarily, the trunking of enclosed belt conveyors is of modular construction, consisting of identical rectangular sections 10S that are bolted together end to end along end flanges 10f (see Fig. 3 and 4). The covers 19 of each section 10S of the trunking are fastened to the upper edges of the side walls by clips 19c.

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An endless belt 20 is trained at the tail end 12 around a tail pulley (not shown), which is supported by bearings 22, and at the head end by a driven head pulley 24 (not shown), which is supported by bearings 26 and is driven by the motor and drive 16 and 18. The belt 20 carries material M (see Fig. 2) supplied at the tail end in a suitable manner along an upper run, along which the belt is supported by spool-like troughing idlers 28 that are journaled in bearings 30 installed outside the side walls 32 of the trunking 10. The tension in the belt is adjusted by an adjuster section 34. The belt 20 returns to the tail end 12 by sliding along the floor 36 of the trunking, which is lined with liner sheets 38 of a rigid low friction polymeric material, such as ultra high molecular weight polyethylene. The liner sheets 38 are substantially coextensive with the trunking floor, except for small gaps where the sheets meet end to end. One or two liner sheets are provided in each section of modular trunking. The sliding of the belt along the trunking floor continuously sweeps dust and material that falls from the upper run of the belt 20 back to the tail end 12, where known devices return it to the upper run for conveyance to the discharge location of the conveyor. Various discharge devices are used with enclosed belt

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conveyors, some of which include stationary tripper valves for diverting the flow.

As discussed above, troughing idlers according to the present invention may be used to particular advantage in enclosed belt conveyors, such as the one shown in Fig. 1 and 2. The troughing idlers of the present invention are supported solely from the side wall 32 of the trunking.

In the embodiment shown in Figs. 3 and 4, a medial roller 40 in the form of a tubular shell is affixed to a horizontal shaft 42, the end portions of which pass through the side walls 32 of the troughing section and seals 43 and are supported by antifriction bearings 44 that are fastened to the side walls. A side roller unit 46 is mounted in each side wall 32. Each unit 46 includes a side roller carrier 48 that has a spindle 50, a hub 52, and inboard and outboard antifriction bearings 54 and 56 interposed between the spindle and the hub. The side roll carrier is attached to the trunking wall 32 by a bracket 57 and supports a side roller 58 in cantilevered relation with respect to the trunking side wall 32 for rotation about the axis of the spindle 50. The medial roller 40 and the side rollers 58 are arranged relative to each other so as to support the belt 20 with side portions 20s sloping upwardly and outwardly with respect to a medial portion 20m. The axes of

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rotation of the side rollers 58 lie in a vertical plane perpendicular to the longitudinal axis of the trunking section 10S and slope downwardly and inwardly from the side walls 32 at an angle that may vary within a wide range, a range of from 20° to 45° being common. The side rollers 58 are spaced apart a small distance longitudinally from the medial roller 40 (see Fig. 4), which allows the lower ends of the side rollers 58 to overlap the end portions of the medial roller 40 in end elevation, thus enabling the belt to be fully supported across its lateral extent.

The side roller unit 100 shown in Fig. 5 has a very short spindle 102 that is attached to a bracket 104 by a nut 106 that is threaded onto the threaded outboard end of the spindle. The bracket 104 has a circular cylindrical tubular side wall 104s, which is vented through a vent slot 104v so that clean air surrounds the hub and bearings for a safe environment. A mounting plate 104m with an elliptical hole receives and is affixed to the side wall 104s of the bracket and is bolted (bolts not shown) to the trunking side wall 32.

The spindle 102 carries a hub 108, which has an internal rib 110 to form shoulders for the outer races of an inboard antifriction bearing 112 and an outboard antifriction bearing 114. The inner races are received on

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the spindle 102 and held in place between a shoulder 102s on the spindle and a retainer nut/washer 116 threaded onto the threaded inboard end of the spindle. The inboard end of the hub 108 lies inboard of the inboard bearing 112 and receives a hub cap 118, which captures a lubricant supplied through a lubricant passage 120 and grease fitting 122 within the hub, prevents particles from escaping from within the hub (e.g., hot pieces of a failed bearing), and keeps dust out of the hub. A grease seal 124 is received between the outboard end of the hub 108 and the spindle 102.

The hub 108 is press-fit (or otherwise suitably connected) to a side roller shell 126. A dust seal 128 keeps dust from within the trunking 10 from escaping from the trunking into the annular space between the shell 126 and the side wall 104s of the bracket. A dust cap 130 keeps dust from entering and collecting inside the shell.

A stationary spindle may be of any desired length, as the side roller unit 200 of Fig. 6 shows. The unit 200 is similar to the unit 100 of Fig. 5. Hence, the same reference numerals as used in Fig. 5, but increased by 100, are applied to Fig. 6, and the description of the unit 100 of Fig. 5 applies in most respects to the unit 600 of Fig. 6. One will observe a different form of mounting bracket 204, the more widely spaced antifriction bearings 212 and

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214, a dust seal 228 secured to the bracket, and the connection of the hub 208 to the roller shell 226 by annular rings 232 and 234.

Fig. 7 shows a side roll unit 300 (the same reference numerals are used in Fig. 7 as in Fig. 5, but increased by 200) in which the hub 308 is affixed to the outboard side of the bracket 304, such as by bolting side flanges (not shown) on the hub to the bracket, and is thus stationary, and the spindle 302 is affixed to the roller shell 336. The structure and function of the unit 300 can be fully understood from the foregoing description of the unit 100 of Fig. 5 and the drawing figure.

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